Comparison of nonlocal approaches in continuum damage mechanics

Citation for published version (APA):

DOI:
10.1016/0045-7949(94)00501-S

Document status and date:
Published: 01/01/1995

Document Version:
Publisher’s PDF, also known as Version of Record (includes final page, issue and volume numbers)

Please check the document version of this publication:

- A submitted manuscript is the version of the article upon submission and before peer-review. There can be important differences between the submitted version and the official published version of record. People interested in the research are advised to contact the author for the final version of the publication, or visit the DOI to the publisher’s website.
- The final author version and the galley proof are versions of the publication after peer review.
- The final published version features the final layout of the paper including the volume, issue and page numbers.

Link to publication

General rights
Copyright and moral rights for the publications made accessible in the public portal are retained by the authors and/or other copyright owners and it is a condition of accessing publications that users recognise and abide by the legal requirements associated with these rights.

- Users may download and print one copy of any publication from the public portal for the purpose of private study or research.
- You may not further distribute the material or use it for any profit-making activity or commercial gain
- You may freely distribute the URL identifying the publication in the public portal.

If the publication is distributed under the terms of Article 25fa of the Dutch Copyright Act, indicated by the “Taverne” license above, please follow below link for the End User Agreement:
www.tue.nl/taverne

Take down policy
If you believe that this document breaches copyright please contact us at:
openaccess@tue.nl
providing details and we will investigate your claim.

Download date: 14. Sep. 2020
COMPARISON OF NONLOCAL APPROACHES IN CONTINUUM DAMAGE MECHANICS

J. H. P. de Vree, W. A. M. Brekelmans and M. A. J. van Gils
Faculty of Mechanical Engineering, University of Technology, P.O. Box 513, 5600 MB Eindhoven, The Netherlands

(Received 4 December 1993)

Abstract—A local continuum damage theory and two distinct nonlocal variants are applied to model the failure behaviour of a construction made of macroscopically brittle material. In the nonlocal formulations a material characteristic length parameter is introduced associated with the width of the microstructural damage zone. The numerical implementation of the approaches has been performed in a finite element code. Simulation results calculated for a plane stress configuration are compared. The local approach solutions show severe lack of mesh objectivity, whereas both the nonlocal solutions converged after mesh refinement. By adequate tuning of the nonlocal descriptions mutually similar responses can be obtained, although intrinsic differences are present in the resulting damage distributions.

1. INTRODUCTION

Continuum damage approaches offer the possibility to simulate the mechanical behaviour of constructions of history-dependent material, irreversibly degenerating under mechanical loads. The damage process is characterized by the development, growth and coalescence of microdefects leading to the formation and propagation of microcracks and eventually to rupture. Microstructural observations are useful for the identification and quantification of history parameters associated with damage[1]. The introduction of damage parameters is adopted from Kachanov [2], who first applied the damage concept to model fracture in creep. Although damage parameters are conceptually related to the observable local deterioration of the material, generally these parameters are phenomenologically incorporated into the constitutive description. In this paper, brittle material behaviour will be considered. Elastic moduli are influenced by damage development. The damage evolution is assumed to be isotropic; explicit time dependence and thermal effects are excluded from the considerations.

The increase of damage generally leads to local softening behaviour: the tangential stiffness becomes negative. It has been shown[3, 4] that a local approach to softening phenomena may lead to a physically unacceptable localization of the deformation; in a finite element context serious mesh sensitivity occurs[5–7]. To overcome the deficiencies of the classical local modelling a number of basically different approaches can be distinguished:

- nonlocal modelling of the constitutive behaviour [4, 8, 9];
- gradient dependent material description [4, 10];
- micropolar (Cosserat) continuum theory [4];
- viscous regularization [11, 12];
- local manipulation of material properties depending on the element size [11].

Gradient dependent models as well as micropolar continuum theory require radical modifications of ordinary finite element codes. It is obvious that viscous regularization is inappropriate for typically brittle material behaviour. The adaptation of material properties is disputable from a physical point of view: global responses may reasonably be predicted, however, calculated local strain and damage distributions will certainly not be in agreement with reality. In this paper two different nonlocal theories are examined and the major structure of the finite element implementation will be explained. The numerical results of a characteristic example are reported and discussed in conclusion.

2. THE LOCAL DAMAGE APPROACH

2.1. General concepts

The influence of isotropic damage on elastic material behaviour is described by the so-called damage parameter $D$. This monotonously increasing scalar quantity, $0 < D < 1$, expresses the level of material degradation [13–16]. Undamaged material is characterized by $D = 0$, the complete loss of stiffness and coherence by $D = 1$. According to the classical continuum damage mechanics approach the preposition of the principle of effective stress [13, 14] leads to the constitutive relationship

$$\sigma = (1 - D)He \quad (1)$$
with \( \sigma \) the column with Cauchy stress components, \( \epsilon \) the column with strain components and with \( H \) Hooke’s elasticity matrix. The product \((1-D)H\) can be considered as the effective elasticity matrix. Implicitly it is assumed that Poisson’s ratio \( v \) is not affected by damage.

The damage evolution is determined by a scalar measure of the strain components: the damage equivalent strain \( \epsilon_{eq} = \epsilon_{eq}(\epsilon) \geq 0 \). There are several alternatives [17, 18] to define \( \epsilon_{eq} \), weighting the strain components differently. A suitable selection has to be made in accordance with material features. In this paper a modification (adopted from Ref. [19]) of the von Mises equivalent strain will be applied:

\[
\epsilon_{eq} = \frac{\gamma - 1}{2\gamma(1-2\nu)} J_1 + \frac{1}{2\gamma} \sqrt{\frac{\gamma - 1}{1-2\nu} J_1^2 + \frac{12\gamma}{(1+\nu)^2} J_2^2} (2)
\]

with \( \gamma \) denoting the ratio of tensile and compressive strength and with \( J_1 \) and \( J_2 \) strain invariants defined by

\[
J_1 = \epsilon_{xx} + \epsilon_{yy} + \epsilon_{zz} \\
J_2 = \frac{1}{2} (\epsilon_{xx}^2 + \epsilon_{yy}^2 + \epsilon_{zz}^2 - \epsilon_{xx} \epsilon_{yy} - \epsilon_{yy} \epsilon_{zz} - \epsilon_{zz} \epsilon_{xx} - 3(\epsilon_{y}^2 + \epsilon_{z}^2 + \epsilon_{x}^2)). (3)
\]

The subscripts \( x, y, z \) in eqn (3) indicate the (strain) components in an orthonormal coordinate system. Increase of damage is only possible if actually \( \epsilon_{eq} \) equals an evolving threshold value \( \kappa \geq 0 \) and increases. The damage remains constant if \( \epsilon_{eq} \) is smaller than \( \kappa \) or if \( \epsilon_{eq} \) is not increasing. These statements are mathematically formulated by

\[
\dot{D} < 0 \quad \text{if} \quad \epsilon_{eq} = \kappa \quad \text{and} \quad \text{if} \quad \epsilon_{eq} > 0 \\
\dot{D} = 0 \quad \text{if} \quad \epsilon_{eq} < \kappa \quad \text{or} \quad \epsilon_{eq} \leq 0. (4)
\]

Consistency requires that

\[
\kappa = \langle \epsilon_{eq} \rangle \quad \text{if} \quad \epsilon_{eq} = \kappa \\
\kappa = 0 \quad \text{if} \quad \epsilon_{eq} < \kappa (5)
\]

where the so-called McAuley brackets \( \langle \cdot \rangle \) are defined by

\[
\langle x \rangle = \frac{1}{2} (x + |x|). (6)
\]

The nondecreasing threshold strain quantity \( \kappa \) is comparable with the yield stress in the theory of elastoplasticity. Strain tensors satisfying \( \epsilon_{eq} = \kappa \) represent the actual damage surface in strain space.
with \( h[ ] \) the Heaviside function defined by
\[
h[x] = \begin{cases} 
0 & \text{if } x < 0 \\
1 & \text{if } x > 0 
\end{cases}
\] (12)
and with \( \kappa \), the actual threshold value given by
\[
\kappa = \max\{\kappa_{i_{\text{eq}, \text{f}}}, \kappa_{i-1} \kappa_{i} \}
\] (13)
where the subscript \( f \) refers to the final converged state in the previous increment \( (k-1) \). It is remarked that for \( \kappa \), an almost zero matrix is substituted when \( \kappa \) approaches the critical value \( \kappa_{c} \) associated with \( D(K_{c}) = 1 \), to avoid numerical problems.

The column with residual forces \( \gamma_{r} \) in the right hand side of eqn (7) is equal to the difference of the external load \( f_{\text{ext}} \) and the internal forces \( \gamma_{f_{\text{int}}} \) to be determined from the actual stress state \( \sigma \):
\[
\gamma_{r} = f_{\text{ext}} - f_{\text{int}}(\sigma) \quad (14)
\]
The iteration process is terminated when the above unbalance becomes negligibly small.

3. NONLOCAL DAMAGE FORMULATIONS

3.1. General concepts

Two different nonlocal formulations are considered with respect to the applicability in brittle fracture simulations, particularly the regularization of deformation localization. Both procedures will be explained in detail in Sections 3.3 and 3.4. Preliminary to that the common features are presented.

The essential idea of the nonlocal approach is the negligence of the generally accepted principle of local action, stating that, in the absence of temperature effects, the stress in a material point is completely determined by the deformation and the deformation history at that point. The actual internal damage parameter \( D \) figuring in the constitutive eqn (1) is now assumed to be dependent on the strain (and the strain history) in a limited finite area enclosing the particular material point where the stress has to be evaluated. Consequently local strain peaks will always have a certain transfer to the environment and thus prevent the localization of the damage, which in turn has a suppressing effect on the progressive growth of the deformation. The nonlocal constitutive relationship for an arbitrary material point reads:
\[
\sigma = (1 - \bar{D}) H \delta e - \delta \bar{D} H e
\] (15)
where \( \bar{D} \) depends on the distribution of the strain and the strain history in the vicinity of the material point. In a formal expression the quantity \( \bar{D} \) at time \( t \) in a point \( x \) in the surrounding volume \( V(x) \) can be written as:
\[
\bar{D}(x, t) = D(\xi(t), \tau) | \xi \in V(x) ; \tau \leq t \].
(16)
Analogous to eqn (4) the nonlocal damage quantity \( \bar{D} \) is assumed to be expressible in a weighted version \( \bar{\epsilon}_{\text{eq}, \text{f}} \) of the equivalent strain:
\[
\bar{D} > 0 \text{ if } \bar{\epsilon}_{\text{eq}, \text{f}} = \kappa \text{ and if } \bar{\epsilon}_{\text{eq}, \text{f}} > 0
\]
\[
\bar{D} = 0 \text{ if } \bar{\epsilon}_{\text{eq}, \text{f}} < \kappa \text{ or if } \bar{\epsilon}_{\text{eq}, \text{f}} \leq 0,
\] (17)
while the threshold \( \kappa \) satisfies
\[
\kappa = \langle \bar{\epsilon}_{\text{eq}, \text{f}} \rangle \quad \text{if } \bar{\epsilon}_{\text{eq}, \text{f}} = \kappa
\]
\[
\kappa = 0 \quad \text{if } \bar{\epsilon}_{\text{eq}, \text{f}} < \kappa
\] (18)
to maintain a consistent formulation.

3.2. Aspects of the finite element implementation

In a finite element context the weighted damage \( \bar{D} \) in a particular integration point \( P \) can be calculated from the nonlocal equivalent strain \( \bar{\epsilon}_{\text{eq}}(\text{and the history)} \) defined by
\[
\bar{\epsilon}_{\text{eq}}^{\text{ip}} = \frac{1}{\sum_{\text{ip}=1}^{\text{np}}} \sum_{\text{ip}=1}^{\text{np}} w^{\text{ip}} \Delta V^{\text{ip}} \bar{\epsilon}_{\text{eq}}^{\text{ip}}
\]
(19)
with \( \text{ip} \) the index of an integration point in a set of \( P \) surrounding integration points, \( \text{np} \) the total number of integration points in that set, \( \Delta V^{\text{ip}} \) the integration volume associated with integration point \( \text{ip} \) and \( w^{\text{ip}} \) the weighting function to be evaluated at the distance \( r \) from point \( P \) to the integration point. Based on eqn (15), the iterative variation of the stress column, necessary to determine the stiffness matrix, can be formulated as
\[
\delta \sigma = (1 - \bar{D}) H \delta e - \delta \bar{D} H e
\]
\[
= (1 - \bar{D}) H \delta e - \delta \bar{D} \frac{\alpha}{1 - \bar{D}}
\] (20)
while \( \delta \bar{D} \) can be expressed in \( \delta \bar{\epsilon}_{\text{eq}} \) by
\[
\delta \bar{D} = \frac{\partial \bar{D}}{\partial \bar{\epsilon}_{\text{eq}}} \delta \bar{\epsilon}_{\text{eq}}
\] (21)
By incorporation of the relationships eqns (18) and (19) the iterative variation of the nonlocal damage in integration point \( P \) can be written as
\[
\delta \bar{D}^{\text{ip}} = \left( \frac{\partial \bar{D}}{\partial \bar{\epsilon}_{\text{eq}}} \right)^{\text{ip}} h[\bar{\epsilon}_{\text{eq}}^{\text{ip}} - \kappa \eta] h[\bar{\epsilon}_{\text{eq}}^{\text{ip}}] \left( \sum_{\text{ip}=1}^{\text{np}} \frac{\partial \bar{\epsilon}_{\text{eq}}^{\text{ip}}}{\partial \bar{\epsilon}_{\text{eq}}} \delta \bar{\epsilon}_{\text{eq}}^{\text{ip}} \right).
\] (22)
The partial derivative of the nonlocal equivalent strain in point \( P \) to the local equivalent strain in point \( \text{ip} \) reads:
\[
\frac{\partial \bar{\epsilon}_{\text{eq}}^{\text{ip}}}{\partial \bar{\epsilon}_{\text{eq}}} = W^{\text{ip}} \frac{\Delta V^{\text{ip}}}{V^{\text{ip}}}
\] (23)
increment $k$ in iteration step $i$, comparable to the relationships in eqn (10), are found:

$$V_i = (1 - \bar{k})H_i$$

$$W_i = \frac{\bar{k}}{1 - \bar{k}} \left( \frac{\partial D}{\partial \kappa} \right) \epsilon_{\kappa} h [\tau_{\kappa} - \bar{k}]$$

expressed in local quantities only (the superscript $P$, denoting the particular integration point, is omitted in these equations).

3.3. The grid method

Among the variety of methods of averaging [6, 20], in this section a strategy is pursued inspired by the observation that, at microstructural level, many materials have a characteristic volume where the damage distribution is almost uniform [21]. The dimension of this characteristic volume is related to the size of material inhomogeneities as the constituting particles or aggregates.

The grid method to be presented here, is merely adopted from Hall and Hayhurst [21]. However, it will be applied in a modified manner. A regular cell grid overlay is placed over an element mesh. Under mesh refinement the grid remains unchanged. The measures of the cells are equal to the characteristic dimension of the material, indicated by the length parameter $\lambda$. In Fig. 1 a grid with typical cells is shown. Essential for the method is the assumption that in every cell of the grid the nonlocal equivalent strain and consequently the nonlocal damage is considered as constant. The local equivalent strains of all individual integration points within the cell are averaged with uniform weighting. The assignment of element integration points to the cells is performed...
Comparison of nonlocal approaches in continuum damage mechanics

only once at the beginning of the analysis and does not change during load incrementation. As a consequence of the procedure proposed, the calculated nonlocal damage will be discontinuous over the boundaries of adjacent cells.

3.4. The continuous average strain method

The nonlocal equivalent strain in an arbitrary integration point \( P \) can also be evaluated as the weighted average of equivalent strains in all material points within a certain radius. This method has been described by Bazant et al. [8, 22] and by Saourides and Mazars [9]. This method principally leads to a continuous damage field. The nonlocal equivalent strain in a particular point \( P \) is determined with eqn (19) by a summation over the \((nip)\) integration points with distance \( r \) to \( P \) not exceeding the material characteristic length \( \lambda \) (with \( nip \) a known number, depending on the local size of the elements in the relevant vicinity of point \( P \)), while the weighting function \( w \) is supposed to be a function of the distance. For further elaborations \( w(r) \) has been selected as a Gaussian weighting function according to Bazant [8]:

\[
w(r) = e^{-\pi(r^2)\lambda^2}
\]

where \( \lambda \) is a control parameter. Bazant and Pijaudier-Cabot [8] recommended \( \lambda = 2 \) in the case of two

![Fig. 3. Stress-strain relationship.](image)

![Fig. 4. The damage \( D \) as a function of threshold \( \kappa/\kappa_c \).](image)

![Fig. 5. Meshes for the quarter plate.](image)

![Fig. 6. Force-displacement diagram, local modelling.](image)

![Mesh 1: 63 elements](image)

![Mesh 2: 107 elements](image)

![Mesh 3: 163 elements](image)

![Mesh 4: 328 elements](image)

![Mesh 5: 504 elements](image)

![Meshes for the quarter plate.](image)

![Mesh 1: 63 elements](image)

![Mesh 2: 107 elements](image)

![Mesh 3: 163 elements](image)

![Mesh 4: 328 elements](image)

![Mesh 5: 504 elements](image)

![Meshes for the quarter plate.](image)

![Mesh 1: 63 elements](image)

![Mesh 2: 107 elements](image)

![Mesh 3: 163 elements](image)

![Mesh 4: 328 elements](image)

![Mesh 5: 504 elements](image)

![Meshes for the quarter plate.](image)

![Mesh 1: 63 elements](image)

![Mesh 2: 107 elements](image)

![Mesh 3: 163 elements](image)

![Mesh 4: 328 elements](image)

![Mesh 5: 504 elements](image)

![Meshes for the quarter plate.](image)

![Mesh 1: 63 elements](image)

![Mesh 2: 107 elements](image)

![Mesh 3: 163 elements](image)

![Mesh 4: 328 elements](image)

![Mesh 5: 504 elements](image)

![Meshes for the quarter plate.](image)

![Mesh 1: 63 elements](image)

![Mesh 2: 107 elements](image)

![Mesh 3: 163 elements](image)

![Mesh 4: 328 elements](image)

![Mesh 5: 504 elements](image)

![Meshes for the quarter plate.](image)

![Mesh 1: 63 elements](image)

![Mesh 2: 107 elements](image)

![Mesh 3: 163 elements](image)

![Mesh 4: 328 elements](image)

![Mesh 5: 504 elements](image)

![Meshes for the quarter plate.](image)

![Mesh 1: 63 elements](image)

![Mesh 2: 107 elements](image)

![Mesh 3: 163 elements](image)

![Mesh 4: 328 elements](image)

![Mesh 5: 504 elements](image)

![Meshes for the quarter plate.](image)

![Mesh 1: 63 elements](image)

![Mesh 2: 107 elements](image)

![Mesh 3: 163 elements](image)

![Mesh 4: 328 elements](image)

![Mesh 5: 504 elements](image)

![Meshes for the quarter plate.](image)

![Mesh 1: 63 elements](image)

![Mesh 2: 107 elements](image)

![Mesh 3: 163 elements](image)

![Mesh 4: 328 elements](image)

![Mesh 5: 504 elements](image)

![Meshes for the quarter plate.](image)

![Mesh 1: 63 elements](image)

![Mesh 2: 107 elements](image)

![Mesh 3: 163 elements](image)

![Mesh 4: 328 elements](image)

![Mesh 5: 504 elements](image)

![Meshes for the quarter plate.](image)

![Mesh 1: 63 elements](image)

![Mesh 2: 107 elements](image)

![Mesh 3: 163 elements](image)

![Mesh 4: 328 elements](image)

![Mesh 5: 504 elements](image)

![Meshes for the quarter plate.](image)

![Mesh 1: 63 elements](image)

![Mesh 2: 107 elements](image)

![Mesh 3: 163 elements](image)

![Mesh 4: 328 elements](image)

![Mesh 5: 504 elements](image)

![Meshes for the quarter plate.](image)

![Mesh 1: 63 elements](image)

![Mesh 2: 107 elements](image)

![Mesh 3: 163 elements](image)

![Mesh 4: 328 elements](image)

![Mesh 5: 504 elements](image)

![Meshes for the quarter plate.](image)

![Mesh 1: 63 elements](image)

![Mesh 2: 107 elements](image)

![Mesh 3: 163 elements](image)

![Mesh 4: 328 elements](image)

![Mesh 5: 504 elements](image)

![Meshes for the quarter plate.](image)
material description has been adopted from related research in Ref. [23]. The one dimensional stress-strain relationship (tension) is bilinear as illustrated in Fig. 3. This bilinear material behaviour matches with the local constitutive modelling in Section 2 if $D = D(\kappa)$ is specified by

$$D(\kappa) = 1 - \frac{\kappa_0 (\kappa - \kappa_0)}{\kappa (\kappa - \kappa_c)}$$

(28)

with $\kappa_0$ and $\kappa_c$ the initial and the critical values of $\kappa$, respectively. Equation (28) is visualized in Fig. 4. The material data [23] used for the numerical calculations are: $E = 6000$ MPa, $\nu = 0.3$, $\gamma = 10$, $\kappa_0 = 2 \times 10^{-2}$, $\kappa_c = 2.7 \times 10^{-2}$ and $x = 10$ mm. The nonlocal relationship $\tilde{B} = \tilde{B}(\kappa)$, required for elaboration of the approaches in Section 3 is taken as equivalent to eqn (28).

The quarter of the plate to be analysed is subdivided into isoparametric four-node plane stress elements. In order to study the mesh dependence, five different meshes (Fig. 5) are applied to discretize the displacement field. The element size varies from large to small in the region of the plate where high gradients of the strain and consequently of the damage are to be expected.

5. RESULTS

5.1. Local modelling

The response results of the analysis with the local damage concept are presented in Fig. 6. The variable along the horizontal axis is the prescribed displacement of the upper edge of the quarter plate model and along the vertical axis the associated total tensile force. Typical differences in the distributions of the damage, determined with a coarse and with a fine mesh, are visualized in Fig. 7 for a tensile force equal to 40 N in the post-peak softening regime. The objectionable consequence of local modelling is obvious. Mesh refinement leads to a vanishing energy dissipation and the deformation localizes in an area with decreasing volume.

5.2. Nonlocal modelling

Figure 8 displays the responses calculated with the grid method and the continuous average strain method using Gaussian weighting with $x = 2$, see eqn (27). It can be observed that for both methods the results converge with refinement of the mesh. However, these two nonlocal regularization procedures predict quite different responses. In Figs 9 and 10 the damage distributions are presented, again for a
Comparison of nonlocal approaches in continuum damage mechanics

**Fig. 9.** Damage distribution, grid method.

**Fig. 10.** Damage distribution, average strain method.

A tensile force of 40 N on the softening path. Especially in Fig. 10 it can be observed that for a coarse mesh the damage distribution already closely resembles the distribution determined with the finest mesh. Based on the considerations of dissipated energy, the results of the grid method can be accessed as rather satisfying; therefore these results are accepted as a reference and it is examined for which value of the weighting parameter $\alpha$ in eqn (27) the average strain method produces a similar response to the grid method. Figure 11 gives the responses calculated with the average strain method for variations of the parameter $\alpha$. Obviously the responses from the grid method and the average strain method approximately coincide for $\alpha = 4$. In Fig. 12 the damage distribution resulting from the average strain method for $\alpha = 4$ is displayed.

**6. CONCLUSIONS AND DISCUSSION**

Continuum damage theories lead to powerful methods for the numerical analysis of the behaviour of softening materials, provided that adequate precautions are taken to limit the localization of the deformation and consequently of the damage. The problems of the mesh objectivity of finite element solutions and correct representation of size effects are intrinsically related. With the application of distinguishable nonlocal approaches in continuum damage mechanics, mesh objective mesh results were obtained in this paper for brittle material behaviour.

**Fig. 11.** Force-displacement diagrams for different values of $\alpha$, average strain method.

**Fig. 12.** Damage distribution with $\alpha = 4$, average strain method.
For the grid method and the weighted average strain method convergence occurred with mesh refinement. These methods can easily be implemented into a classical finite element computer code; the increase of CPU-time needed for the nonlocal calculations proposed is small. Manipulations in the continuous weighted average strain method to produce appropriate results, leads to a value of approximately 4 for the weighting parameter $x$. This value is not in accordance with the recommendation in the literature [8], originating from mathematical considerations. This deviation, here actually established for a particular example, should be examined in a more general context in the continuation of this research.

Apart from the availability of sufficient material data to describe the local softening phenomena, a number of suitable choices have to be made to achieve a proper simulation of damaging configurations. Using the theory presented in this paper, the following items have to be addressed carefully:

- the definition of the equivalent strain; every material has its own characteristic internal disintegration mechanism associated with some combination of strain components. Therefore a variety of equivalent strain definitions is known in the literature leading to different simulation results. The correct choice to simulate experimental observation can only be accessed by hybrid numerical and experimental research;
- the particular quantity in the nonlocal approach that will be averaged; in this paper averaging of the equivalent strain has been elaborated. In the literature also other choices have been made [21];
- the averaging method; two essentially different averaging procedures have been considered in this paper, leading to a discontinuous and a continuous description of damage evolution. Microstructural investigations should give evidence of the physical relevance of one of these approaches for a particular material. The weighted average strain method offers a number of possibilities for tuning the modelling description on physical experience by adjusting the weighting function [24].

Acknowledgement — The results presented in this paper have been obtained by implementation of the theoretical developments into the DIANA finite element code (DIANA Analysis by, Delft, the Netherlands). The authors gratefully acknowledge D. G. Roddeman (from T.N.O. Building and Construction Research, Delft, the Netherlands) for his support.

REFERENCES