Nonlinear disturbance decoupling and linearization: a partial interpretation of integral feedback

Citation for published version (APA):

DOI:
10.1109/CDC.1993.325883

Document status and date:
Published: 01/01/1993

Document Version:
Publisher’s PDF, also known as Version of Record (includes final page, issue and volume numbers)

Please check the document version of this publication:
• A submitted manuscript is the version of the article upon submission and before peer-review. There can be important differences between the submitted version and the official published version of record. People interested in the research are advised to contact the author for the final version of the publication, or visit the DOI to the publisher’s website.
• The final author version and the galley proof are versions of the publication after peer review.
• The final published version features the final layout of the paper including the volume, issue and page numbers.

Link to publication

General rights
Copyright and moral rights for the publications made accessible in the public portal are retained by the authors and/or other copyright owners and it is a condition of accessing publications that users recognise and abide by the legal requirements associated with these rights.
• Users may download and print one copy of any publication from the public portal for the purpose of private study or research.
• You may not further distribute the material or use it for any profit-making activity or commercial gain
• You may freely distribute the URL identifying the publication in the public portal.

If the publication is distributed under the terms of Article 25fa of the Dutch Copyright Act, indicated by the “Taverne” license above, please follow below link for the End User Agreement:
www.tue.nl/taverne

Take down policy
If you believe that this document breaches copyright please contact us at:
openaccess@tue.nl
providing details and we will investigate your claim.

Download date: 20. Aug. 2020
Nonlinear disturbance decoupling and linearization: a partial interpretation of integral feedback

H.C.J. Huijberts
Dept. of Mathematics and Comp. Sci.
Eindhoven Univ. of Technology
P.O. Box 513
5000 MB Eindhoven
The Netherlands.

H. Nijmeijer A.C. Ruiz
Dept. of Appl. Mathematics
Univ. of Twente
P.O. Box 217
7500 AE Enschede
The Netherlands

Abstract
The relation between the solvability of the disturbance decoupling problem for a nonlinear system and its linearization around a working point is investigated. It turns out that generally the solvability of the disturbance decoupling via regular dynamic state feedback is preserved under linearization. This result gives a partial interpretation of introducing integral action in classical PID control applied to nonlinear systems. The theory is illustrated by means of a worked example.

1. Introduction
Like in the linear geometric theory, [16], one of the first "structural" synthesis problems that has been posed and has been solved locally in a nonlinear context, is the so-called disturbance decoupling problem (DDP) for a nonlinear system. This problem may be stated as follows. Consider the nonlinear control system Σv described by

\[
\begin{align*}
\dot{x} &= f(x) + g(x)u + p(x)q \\
y &= h(x)
\end{align*}
\]  

(1)

where \( x = \text{col}(z_1, \ldots, z_n) \in \mathbb{R}^n \) are local coordinates for the state space manifold \( X \), \( u \in \mathbb{R}^m \) denotes the controls, \( q \in \mathbb{R}^q \) the disturbances and \( y \in \mathbb{R}^p \) the outputs. Let \( g_1, \ldots, g_m \) denote the columns of the matrix \( g \) and let \( p_1, \ldots, p_r \) denote the columns of the matrix \( p \). All data in (1) are local coordinates for the state space manifold \( X \), \( u \in \mathbb{R}^m \) denotes the controls, \( q \in \mathbb{R}^q \) the disturbances and \( y \in \mathbb{R}^p \) the outputs. Let \( g_1, \ldots, g_m \) denote the columns of the matrix \( g \) and let \( p_1, \ldots, p_r \) denote the columns of the matrix \( p \). All data in (1), i.e., the vector fields \( f, g_1, \ldots, g_m \) and \( p_1, \ldots, p_r \) as well as the function \( h \), will be assumed to be analytic in this paper. In the DDP one is asked to design, if possible, a static state feedback

\[
\begin{align*}
Q_s : u &= \alpha(x) + \beta(x)v \\
& \quad \text{with } \alpha(x) \text{ and } \beta(x) \text{ respectively an } m \text{-vector and an } (m, m) \text{-matrix depending analytically on } x, \text{ and where } v \in \mathbb{R}^m \text{ denotes a new control vector, such that in the closed loop system } \Sigma_v \circ Q_s \text{ the output } y \text{ is unaffected by the disturbances } q, \text{ no matter how } v \text{ is chosen. Usually the DDP is considered under the assumption that in the static feedback law (2) the matrix } \beta(x) \text{ is nonsingular for all } v \text{, this in order to keep as much control on the system as possible, while at the same time disturbance decoupling is achieved. Define the distributions } \mathcal{G} := \text{span}(g_1, \ldots, g_m), \mathcal{P} := \text{span}(p_1, \ldots, p_r) \text{ and let } \mathcal{D}^* \text{ be the maximal locally}
\end{align*}
\]  

controlled invariant distribution in Kerdh for (1). If the distributions \( \mathcal{G}, \mathcal{D}^* \subset \mathcal{G} \) have constant dimension, the DDP is locally solvable for (1) if and only if \( \mathcal{P} \subset \mathcal{D}^* \) (cf. [11],[12]).

Another version of the disturbance decoupling problem is the so-called dynamic disturbance decoupling problem, abbreviated DDP, in which the disturbance decoupling is done via a dynamic state feedback. In other words, define a dynamic state feedback as

\[
\begin{align*}
Q_d : \begin{cases}
\dot{x} &= \alpha(x, z) + \beta(x, z)v \\
u &= \gamma(x, z) + \delta(x, z)v
\end{cases}
\end{align*}
\]  

(3)

with the compensator state \( z \in \mathbb{R}^n \) and \( v \in \mathbb{R}^m \) is again a new control vector. We assume the system (3) to be regular, which implies invertibility between the old controls \( u \) and the new controls \( v \) of this system (see Section 3 for a more specific definition). In the DDDP one seeks a dynamic state feedback (3) such that in the closed loop system \( \Sigma_v \circ Q_d \) the disturbances \( q \) do not influence the outputs. The DDDP has been posed and solved in a local fashion in [5],[6] (see also [13]) and one of the remarkable conclusions is that for nonlinear systems the DDDD might be solvable for systems for which the (regular) DDP is not. The latter statement, as is known (cf. [1]), in contrast with the linear theory, since for linear systems for which the DDP is not solvable, also the DDDP is not solvable.

The purpose of the present paper is to make a further step in exploiting the ideas of using dynamic feedback in achieving disturbance decoupling while at the same time trying to relate this to one of the basic approaches in control engineering practice, namely linearization. Assume for the moment that (1) is considered around some working point \( x_0 \), so \( f(x_0) = 0 \), and let the (Jacobian) linearization \( \Sigma_{v^0} \) of (1) around \( x_0 \) be given by

\[
\begin{align*}
\dot{\xi} &= F\xi + Gu + Pq \\
\eta &= H\xi
\end{align*}
\]  

(4)

A first elementary engineering approach to tackle any synthesis problem for (1) would be to address the same design goal for its linearization (4) and use the linear solution as an approximate solution for the nonlinear system.

We will show that for a large (generic) class of nonlinear systems (1) this approach indeed makes sense in case one allows regular dynamic state feedbacks in the solution of the disturbance decoupling problem for the linear system (4). As mentioned before, for the linear system (4) itself
it would be enough to limit ourselves to static state feedbacks for solving the DDP (if the problem is solvable at all), but we will show explicitly that only the solvability of the DDDP is preserved under linearization.

The solution of the DDDP we propose in our papers [5],[6],[7] is very much based on a special class of dynamic state feedbacks, that we call Singh compensators. A particular feature of such a compensator is that some (but not all) of the controls \( u_i \) are determined a certain number of times. Thus we encounter in a Singh compensator schemes as \( z_i = u_i, u_i = z_i \), which is the same as allowing \( u_i = v_i \). Since this type of dynamic compensation will naturally arise in our solution of the DDDP, one could view our results as a partial justification of introducing integral action in classical PID-control applied to nonlinear systems.

The paper is organized as follows. In Section 2 we derive a connection between the solvability of the static disturbance decoupling problem for a nonlinear system and its linearization around a working point. Since this result is only included to be used in Section 3, we will not give it in its full generality but only for the specific case that the decoupling matrix of the nonlinear system under consideration is invertible. In Section 3 we first introduce some algebraic preliminaries and a special sort of regular dynamic state feedback, the Singh compensator. After this, we derive our main result. In Section 4 a worked example is given. Finally, in Section 5 some conclusions are drawn. Throughout the paper we restrict ourselves to square systems, i.e., \( m = p \) in (1). All the results in the paper can be easily extended to nonsquare systems.

2. Static disturbance decoupling and linearization

In this section we investigate the connection between the solvability of the DDP for \( \Sigma_q \) around an equilibrium point and the solvability of the problem for the linearization of \( \Sigma_q \) around this equilibrium point. We restrict ourselves to the case that the decoupling matrix of \( \Sigma_q \) has full rank at the equilibrium point.

The decoupling matrix of \( \Sigma_q \) may be defined in the following way. Let \( \Sigma_0 \) denote the system \( \Sigma_q \) without disturbances, i.e., \( q \equiv 0 \). For \( \Sigma_0 \), we define inductively (with \( y^{(0)} := y \))

\[
y^{(k+1)}(x, u, \cdots, u^{(k)}) = \sum_{i=0}^{k} \frac{\partial_y^{(k)}}{\partial u^i} y^{(k)} + \frac{\partial_y^{(k)}}{\partial x} f(x) + g(x) u + \frac{\partial y^{(k)}}{\partial u} u^{(k+1)}
\]

The relative degree \( r_i \) of \( y_i \) (\( i = 1, \cdots, m \)) is defined by

\[
r_i = \min\{k \in \mathbb{N} \mid \frac{\partial y^{(k)}}{\partial u^{i}} \neq 0\}
\]

Assume that all relative degrees of \( \Sigma_q \) are finite. Then we may define the decoupling matrix \( A(x) \) ([11],[12]) of \( \Sigma_q \) by

\[
A(x) = \begin{pmatrix} \frac{\partial y_i^{(r_i)}}{\partial u_j} \end{pmatrix}_{i,j=1,\cdots,m}
\]

Let \( x_0 \in \mathcal{X} \) be an equilibrium point of \( \Sigma_q \), i.e., \( f(x_0) = 0 \). Assume (without loss of generality) that \( h(x_0) = 0 \). Let \( L \Sigma_q \), the linearization of \( \Sigma_q \) around \( x_0 \), be given by (4). We make the following assumptions.

Assumption 2.1 The decoupling matrix \( A(x) \) of \( \Sigma_q \) is invertible at \( x = x_0 \).

Assumption 2.2 The distributions \( \mathcal{P} \) and \( \Delta^* \cap \mathcal{P} \) have constant dimension on a neighborhood of \( x_0 \).

We then have the following result (see [10]).

Theorem 2.3 Consider the system \( \Sigma_q \) and let \( x_0 \in \mathcal{X} \) be an equilibrium point of \( \Sigma_q \) satisfying \( h(x_0) = 0 \) and Assumptions 2.1 and 2.2. Then the DDP for \( \Sigma_q \) is locally solvable around \( x_0 \) if and only if it is solvable for \( L \Sigma_q \).

Assumption 2.2 and Theorem 2.3 may be interpreted as follows. Consider (5) and the system \( \Sigma_q \). Then we may define, analogously to (6), disturbance relative degrees \( s_i \) (\( i = 1, \cdots, m \)) for \( \Sigma_q \) by

\[
s_i = \min\{k \in \mathbb{N} \mid \frac{\partial y_i^{(k+1)}}{\partial x} p(x) \neq 0\} \quad (i = 1, \cdots, m)
\]

Thus \( s_i \) is the smallest time derivative of \( y_i \) that explicitly depends on the disturbances. It is well known (see [12]) that if Assumption 2.1 holds, the DDP is solvable for \( \Sigma_q \) if and only if \( r_i < s_i \) (\( i = 1, \cdots, m \)). In case Assumption 2.1 does not hold, this is only a necessary condition for solvability of the DDP (see [12]).

Denote the relative degrees of \( L \Sigma_q \) by \( r_i' \) (\( i = 1, \cdots, m \)) and its disturbance relative degrees by \( s_i' \) (\( i = 1, \cdots, m \)). Furthermore, let the decoupling matrix of \( L \Sigma_q \) be denoted by \( A_i' \). It can then be shown that \( r_i' \geq r_i \), \( s_i' \geq s_i \) (\( i = 1, \cdots, m \)). Assuming that \( A_i' \) is invertible, we have that the DDP is solvable for \( L \Sigma_q \) if and only if \( r_i' < s_i' \) (\( i = 1, \cdots, m \)). It is then clear that from this we can conclude solvability of the DDP for \( \Sigma_q \) if \( r_i' = r_i \), \( s_i' = s_i \) (\( i = 1, \cdots, m \)). Now, if Assumption 2.1 holds, it follows immediately from \( r_i = s_i \) that \( r_i' = s_i' \), if Assumption 2.2 is satisfied. Thus Assumption 2.2 is posed in order to guarantee the coincidence of the (disturbance) relative degrees of \( \Sigma_q \) and \( L \Sigma_q \).

3. Dynamic disturbance decoupling and linearization

In this section we generalize the results from Section 2 to the case of disturbance decoupling via regular dynamic state feedback. We restrict ourselves to systems of full rank. From the discussion at the end of Section 2 it follows that if we can find a dynamic state feedback for the system under consideration such that the decoupling matrix of the compensated system is invertible and the relative degrees of the compensated system are smaller than the disturbance relative degrees of the compensated system, then the DDP for the compensated system is solvable and so the DDDP for the original system is solvable. So in fact what we need to do in order to solve the DDP, is to find a dynamic state feedback that satisfies the first requirement and at the same time assures that the relative degrees of the compensated system remain as small as possible. In Section 1.1 such a special dynamic state feedback, the Singh compensator, is introduced. It has the
property that it is a dynamic state feedback of minimal dimension (\( \nu \) in (3) is as small as possible) that satisfies the first requirement above. The relative degrees of the compensated system are intrinsically defined: they are the so called essential orders (\( [3] \)) of the original system (cf. [7]). The connection between the solvability of the DDDP for the original system and its linearization around an equilibrium point is then established in Section 3.2 by spelling out assumptions that guarantee that the (disturbance) relative degrees of a system plus Singh compensator equal the (disturbance) relative degrees of the linearization of system plus Singh compensator around an equilibrium point. Here we use the special properties of a Singh compensator with respect to linearization that were reported in [9].

3.1 Mathematical preliminaries and Singh compensator

We start with some algebraic concepts that were introduced in [2]. Consider the nonlinear system \( \Sigma_0 \), i.e., the system derived from \( \Sigma_0 \) by setting \( q \equiv 0 \). Recall that a meromorphic function \( \eta \) is a function of the form \( \eta = \pi/\theta \), where \( \pi \) and \( \theta \) are analytic functions. Assume that the control functions \( u(t) \) are \( n \) times continuously differentiable. Then define \( u^{(i)} := u, u^{(i+1)} := (d/dt)u^{(i)} \). View \( x, u, \ldots, u^{(n-1)} \) as variables and let \( K \) denote the field consisting of the set of rational functions of \( u, \ldots, u^{(n-1)} \) with coefficients that are meromorphic in \( z \). Note that \( y, y_1, \ldots, y^{(m)} \) defined by (5) have components in the field \( K \). Let \( E \) denote the vector space over \( K \) spanned by \( \{dx, dy, \ldots, dy^{(k)}\} \).

\[ E_n = \text{span}_K\{dx, dy, \ldots, dy^{(k)}\} \] (9)

Then the rank \( r^*(\Sigma_0) \) of \( \Sigma_0 \) is defined by (see [2]) \( r^*(\Sigma_0) = \dim_K E_n = \dim_K E_{n-1} \). Note that we always have \( r^*(\Sigma_0) \leq m \). \( \Sigma_0 \) is said to be of full rank if \( r^*(\Sigma_0) = m \).

Now consider a dynamic state feedback \( Q_d \) for \( \Sigma_0 \) of the form (3). \( Q_d \) is said to be a regular dynamic state feedback for \( \Sigma_0 \) if the system

\[
\begin{align*}
\dot{x} &= f(x) + g(x)u \\
\dot{z} &= \alpha(z, x) + \beta(z, x)u \\
u &= \gamma(z, x) + \delta(z, x)u
\end{align*}
\] (10)

with controls \( u \) and outputs \( u \) has full rank (see [2]).

Next we present a special sort of regular dynamic state feedback for \( \Sigma_0 \), that we call a Singh compensator (see [14],[5]). Here we restrict ourselves to a system \( \Sigma_0 \) with the property that \( \Sigma_0 \) is of full rank. The results can easily be extended to systems that do not have full rank (see e.g. [6],[4]). So, consider a nonlinear control system \( \Sigma \) of full rank. For \( r, s \in NV \), introduce the notation \( I_{rs} := \{r, \ldots, s\} \). Using e.g. Singh's algorithm ([14],[2]), we can then find a permutation of the outputs and positive integers \( \gamma_1, \ldots, \gamma_m \) satisfying \( \gamma_1 \leq \cdots \leq \gamma_m \leq n \), such that for \( k = 1, \ldots, n \),

\[
g_k = \bar{\alpha}(x, \{y_{\gamma_i} \mid i \in I_{\gamma_k} \}, j \in I_{\gamma_k})
\] (12)

where the matrices \( B_k := \{\bar{B}_1, \ldots, \bar{B}_s\} \) have full row rank over \( K \) (cf. [2],[7]). Define \( \bar{Y}_n := \{\bar{y}_1, \ldots, \bar{y}_s\} \) and \( \bar{y}_k := (\bar{y}_1, \ldots, \bar{y}_s)^T \). Then (12) yields in particular:

\[
\begin{align*}
\bar{y}_n &= \bar{\Lambda}(x, \{\bar{y}_i \mid i \in I_{\gamma_n} \}, j \in I_{\gamma_n}) + \\
\bar{B}_n(x, \{\bar{y}_i \mid i \in I_{\gamma_n} \}, j \in I_{\gamma_n})u
\end{align*}
\] (13)

We call a pair \((x, y) = (x_0, 0)\) a strongly regular point for \( \Sigma_0 \) if for every possible permutation of the outputs as described above, the matrix \( B_n \) in (13) has full row rank over \( K \), when evaluated at \((x_0, 0)\). If the pair \((x_0, 0)\) is a strongly regular point for \( \Sigma_0 \), we know that for (13) there exists a neighborhood \( U \subset \mathcal{K} \) of \( x_0 \) and a neighborhood \( Y_0 \subset K^m \) of \( \{\bar{y}_1, \ldots, \bar{y}_s\} \) such that \( \bar{B}_n \) is invertible on \( U \times Y_0 \). Then on \( U \times Y_0 \) we obtain from (13):

\[
\begin{align*}
u &= \bar{B}_n^{-1}(\bar{Y}_n - \bar{\Lambda})
\end{align*}
\] (14)

Clearly, \( \gamma_i \) is the lowest time-derivative of \( y_i \), appearing in the right hand side of (14). Let \( \delta_i \) be the highest time-derivative of \( y_i \), appearing in the right hand side of (14). It can be shown that the \( \delta_i \) and \( \sum_{j=1}^{m} \gamma_j \) are intrinsic, i.e., independent of the permutation of the outputs that is chosen (cf. [7]). In fact, the \( \delta_i \) are just the essential orders (\( [5] \)) of \( \Sigma \). Hence also \( \sum_{j=1}^{m} \gamma_j - \gamma_i \) is intrinsic. Moreover, the right hand side of (14) is affine in \( \bar{y}_i \).

Thus we may rewrite (14) as

\[
\begin{align*}
u &= \bar{\phi}(x, \{y_i^{(j)} \mid i \in I_{\gamma_1}, j \in I_{\gamma_1} \}) + \\
&\sum_{i=1}^{m} \phi_2(x, \{y_i^{(j)} \mid i \in I_{\gamma_1}, j \in I_{\gamma_1} \})y_i^{(j)}
\end{align*}
\] (15)

for certain vector-valued functions \( \phi_1, \phi_2 \) \((i = 1, \ldots, m)\). Let \( z_i \) \((i = 1, \ldots, m)\) be a vector of dimension \( \gamma_i - \gamma_i \) and consider the system:

\[
\begin{align*}
\dot{z}_i &= A_i z_i + B_i u_i \quad (i = 1, \ldots, m) \\
u &= \phi_1(z_1, z_1, \ldots, z_m) + \\
&\sum_{i=1}^{m} \phi_2(z_1, z_1, \ldots, z_m)u_i
\end{align*}
\] (15)

with \((A_i, B_i) \) in Brunovsky canonical form. Then (15) is called a Singh compensator for \( \Sigma \) around \( x_0 \).

The Singh compensator has the following properties (see [4],[7],[9]).

**Proposition 3.4.** Consider the nonlinear system \( \Sigma_0 \) and assume that \( r^*(\Sigma_0) = m \). Let \( x_0 \in \mathcal{K} \) be an equilibrium point of \( \Sigma_0 \). Assume that \( h(x_0) = 0 \) and that \((x, y) = (x_0, 0)\) is a strongly regular point for \( \Sigma_0 \). Let \( Q \) be a Singh compensator for \( \Sigma_0 \). Then
(i) \( Q \) is a regular dynamic state feedback for \( \Sigma_q \).

(ii) \( Q \) is a minimal order decoupling compensator for \( \Sigma_q \).

(iii) a. The point \((x, z) = (x_0, 0)\) is an equilibrium point for \( \Sigma_q \) if \( Q \) exists.

b. Denote by \( L(\Sigma_q \circ Q) \) the linearization of \( \Sigma_q \circ Q \) around \((x_0, 0)\). Then \( L(\Sigma_q \circ Q) = L\Sigma_q \circ Q \), where \( L\Sigma_q \), the linearization of \( \Sigma_q \) around \((x_0, 0)\), is a Singh compensator for \( L\Sigma_q \).

c. Conversely, if \( R \) is a Singh compensator for \( L\Sigma_q \), then there is a Singh compensator \( Q \) for \( \Sigma_q \) such that \( LQ = R \) and \( L(\Sigma_q \circ Q) = L\Sigma_q \circ Q \).

3.2 Dynamic disturbance decoupling and linearization

The disturbance decoupling problem via regular dynamic state feedback is defined below.

Definition 3.5 Disturbance decoupling problem via regular dynamic state feedback (DDDP) Consider a nonlinear system \( \Sigma_q \) and let a point \( x_0 \in X \) be given. The DDDP is said to be locally solvable around \( x_0 \) if there exist a regular dynamic state feedback \( Q \) for \( \Sigma_q \) of the form (3), a neighborhood \( U \subset \mathbb{R}^n \) of \( x_0 \) and an open subset \( Z \subset \mathbb{R}^m \) such that the outputs of the composed system \( \Sigma_q \circ Q \) restricted to \( U \times Z \) are independent of the disturbances.

The following theorem, which can be found in [5],[6], gives a local solution of the DDDP. In the statement of the theorem we employ the following notation. In (12) for \( \Sigma_q \), the \( y_k^{(s)} \) \((k = 0, \ldots, n; y_0 = y)\) can be viewed as functions on \( X \subset \mathbb{R}^n \). By the same token, \( \text{Kerd}\gamma_k^{(s)} \)

\((k = 0, \ldots, n)\) defines a distribution on \( X \). Define the following subsets \( G_s, P_s \) on \( X \) by \( G_s := \emptyset \times \{0\} \), \( P_s := P \times \{0\} \). For a particular permutation of the outputs of \( \Sigma_q \) as described in Subsection 3.1, define \( \Delta_x := \cap \text{Kerd}\gamma_k^{(s)} \).

Theorem 3.6 Consider the nonlinear system \( \Sigma_q \) and let \( x_0 \in X \) be such that \((x_0, 0)\) is a strongly regular point for \( \Sigma_q \). Then the DDDP is locally solvable around \( x_0 \) if and only if for every permutation of the outputs for \( \Sigma_q \) as described in Subsection 3.1) we have

\[ P_s \subset \Delta_x \tag{16} \]

Moreover, if the DDDP is locally solvable around \( x_0 \), every Singh compensator for \( \Sigma_q \) around \( x_0 \) solves the DDDP for \( \Sigma_q \).

Remark 3.7 Another way of solving the DDDP can be found in [13].

Consider an equilibrium point \( x_0 \in X \) of \( \Sigma_q \) satisfying \( A(x_0) = 0 \), and the linearization \( LE_x \) of \( \Sigma_q \) around \( x_0 \). We investigate the connection between the solvability of the DDDP for \( \Sigma_q \) and \( LE_x \). The following assumptions are made.

Assumption 3.8 \((x, y) = (x_0, 0)\) is a strongly regular point for \( \Sigma_q \).

Assumption 3.9 For every permutation of the outputs of \( \Sigma_q \) as described in Section 2, \( P_x \) and \( \Delta_x \cap P_x \) have constant dimension on a neighborhood of \((x_0, 0)\) in \( X \).

We now come to the statement of our main result.

Theorem 3.10 Consider the nonlinear system \( \Sigma_q \), where \( \rho'(x_0) = m \). Let \( x_0 \in \mathbb{R}^n \) be an equilibrium point of \( \Sigma_q \) satisfying \( h(x_0) = 0 \) and Assumptions 3.8 and 3.9. Then the DDDP for \( \Sigma_q \) is locally solvable around \( x_0 \) if and only if it is solvable for \( L\Sigma_q \).

Proof (necessity) Assume that the DDDP for \( \Sigma_q \) is locally solvable around \( x_0 \). Let \( Q \) be a Singh compensator that solves the problem around \( x_0 \). Then we have in particular that the DDP is solvable for \( \Sigma_q \circ Q \). It can easily be checked (see e.g. [7]) that the decoupling matrix of \( \Sigma_q \circ Q \) is the same as that of \( \Sigma_q \). Hence by Theorem 2.3 the DDP is solvable for \( L(\Sigma_q \circ Q) \), the linearization of \( \Sigma_q \circ Q \) around \((x_0, 0)\). By Proposition 3.4 we have that \( L(\Sigma_q \circ Q) = L\Sigma_q \circ Q \), where \( R \) is a Singh compensator for \( L\Sigma_q \). Since by Proposition 3.4 \( LQ \) is a regular dynamic state feedback for \( L\Sigma_q \), this means that the DDDP is solvable for \( L\Sigma_q \).

(sufficiency) Assume that Assumptions 3.8 and 3.9 hold and that the DDDP is solvable for \( L\Sigma_q \). Hence by Theorem 2.3 the DDP is solvable for \( L(\Sigma_q \circ Q) \), the linearization of \( \Sigma_q \circ Q \) around \((x_0, 0)\). By Proposition 3.4 there is a Singh compensator \( Q \) for \( \Sigma_q \) such that \((x_0, 0)\) is an equilibrium point of \( \Sigma_q \circ Q \) and such that \( L(\Sigma_q \circ Q) = L\Sigma_q \circ Q \). As above, we have that the decoupling matrix of \( \Sigma_q \circ Q \) is the same as that of \( \Sigma_q \). Then it follows that \( \Delta^*_x \), the maximal locally controlled invariant distribution in \( \text{Kerd}\gamma_x \) for \( \Sigma_q \), is given by (see [11],[12])

\[ \Delta^*_x = \Delta_x \cap \text{Kerd}\gamma_x \tag{17} \]

This implies that Assumption 3.9 for \( \Sigma_q \) is the same as Assumption 2.2 for \( \Sigma_q \). Hence by Theorem 2.3 the solvability of the DDDP for \( L\Sigma_q \) implies the solvability of the DDP for \( \Sigma_q \). Since by Proposition 3.4 the Singh compensator \( Q \) is a regular dynamic state feedback for \( \Sigma_q \), this implies on its turn that the DDDP is solvable for \( \Sigma_q \).

From Theorem 3.10 it follows that if Assumptions 3.8 and 3.9 hold, the solvability of the DDP for \( L\Sigma_q \) implies solvability of the DDDP for \( \Sigma_q \), but not necessarily solvability of the DDP for \( \Sigma_q \) (for a counter example, see the following section). If indeed the DDDP, but not the DDP, is solvable for \( \Sigma_q \), no static state feedback that solves the DDP for \( L\Sigma_q \) will be a first order approximation of a feedback that solves the DDP for \( \Sigma_q \). As a result of this static state feedback will in general not result in a satisfactory disturbance attenuation when applied to \( \Sigma_q \). At the same time the remedy is clear: one should look for a dynamic state feedback that solves the DDP for \( L\Sigma_q \) and that at the same time is the linearization of a dynamic state feedback that solves the DDDP for \( \Sigma_q \).

By Proposition 3.4 and Theorem 3.10, any Singh compensator for \( L\Sigma_q \) will do this job (provided the DDP is solvable for \( L\Sigma_q \)). In other words, one should incorporate integral action to some of the controls.
4. Example

Consider a nonlinear system in $\mathbb{R}^n$ of the form

$$\dot{x}_i = (x_{i+1} + 1)x_i$$

with $u \in \mathbb{R}^n$, $y \in \mathbb{R}^m$, $q$ an unknown disturbance and equilibrium point $x_0 = 0$. For this system we have $\Delta^* = \text{span} \{ \partial q / \partial x_k \}$, and since $P = \text{span} \{ \partial q / \partial x_k \} \not\subset \Delta^*$, the DDP is not solvable for $\Sigma_q$.

The linearization of $\Sigma_q$ around $x_0$ is given by

$$L\Sigma_q = \left\{ \begin{array}{l}
\dot{\xi}_1 = \xi_1 \\
\dot{\xi}_2 = \xi_2 \\
\dot{\xi}_3 = -\xi_3 + \xi_4 + \xi_5 - \xi_6 \\
\dot{\xi}_4 = \xi_4 \\
\dot{\xi}_5 = -15\xi_5 + q \\
\dot{\xi}_6 = 8\xi_6 - 8\xi_4 - 16\xi_6
\end{array} \right\}$$

with $\xi = (\xi_1, \xi_2, \xi_3, \xi_4, \xi_5, \xi_6)^T$.

With equilibrium point $\xi_0 = 0$. After some calculations we find that for $L\Sigma_q$ we have $\nu^* = \text{span} \{ \xi_2, \xi_3, \xi_4 \}$, where $\nu_i$ denotes the $i$-th basis vector of the standard basis of $\mathbb{R}^n$. The DDP for $L\Sigma_q$ is solvable because $\text{Im} P = \text{span} \{ \xi_1 \} \subset \nu^*$. One regular static state feedback which solves the DDP for $L\Sigma_q$ is

$$\begin{align*}
\dot{u}_1 &= -4\xi_1 + \tilde{v}_1 \\
\dot{u}_2 &= -\xi_2 - \xi_3 + \xi_4 + \xi_5 + \xi_6 + \tilde{u}_2,
\end{align*}$$

where $\tilde{v}_1 = (\tilde{v}_1, \tilde{v}_2)^T$ are the new controls. We remark at this point that the regular static state feedback (18) does not correspond to a first order approximation of a static state feedback which approximately solves the local DDP for the system $\Sigma_q$ since, we have seen, the DDP for $\Sigma_q$ is not solvable. An additional evidence of this fact is provided by the following numerical simulations. In these simulations we apply the linear static state feedback (18) to $\Sigma_q$ with the new controls $\tilde{v}_1, \tilde{v}_2$ designed to provide closed loop asymptotic stability.

The initial conditions are chosen to be $x(0) = (0.2, 0.1, 0, 0, 0.6, 0)^T$ and the disturbance $q$ is selected as a sinusoidal function of time: $q(t) = 50 \sin t (0 \leq t \leq 40)$. Figures 1 and 2 show how the outputs of the nonlinear system $\Sigma_q$ are influenced by the disturbance $q(t)$. Now, some calculations will show that $\Sigma_q$ satisfies Assumptions 3.6 and 3.7 and that $\rho(\Sigma_q) = m = 2$. Hence, the DDPD for $\Sigma_q$ is locally solvable around $x_0$ and thus it is also solvable for $\Sigma_q$. A Singh compensator that solves the DDPD for $L\Sigma_q$ is

$$LQ = \left\{ \begin{array}{l}
\dot{x}_1 = \tilde{v}_1 \\
\dot{u}_1 = \tilde{v}_1 \\
\dot{u}_2 = -\xi_2 - \xi_3 + \xi_4 + \xi_5 - \tilde{x}_1 + \tilde{v}_1 + \tilde{v}_2.
\end{array} \right\}$$

In order to have an asymptotically stable closed loop system $L\Sigma_q \circ LQ$ we will set

$$\begin{align*}
\tilde{v}_1 &= -4\xi_1 + \tilde{x}_1 + \tilde{v}_1 \\
\tilde{v}_2 &= -\xi_2 + \xi_3 + 8\xi_6 + \tilde{v}_2.
\end{align*}$$

where $\tilde{v}_1, \tilde{v}_2$ are the new controls. It can be easily checked that the system $LQ \circ L\Sigma_q$ together with the additional feedback (20) remains disturbance decoupled. Since the DDPD is solvable for $\Sigma_q$, we can construct for the system $\Sigma_q$ a Singh compensator to obtain a solution for the DDPD. A Singh compensator is given by

$$Q = \left\{ \begin{array}{l}
\dot{x} = v_1 \\
u_1 = \frac{x}{x^2 + 1} \\
u_2 = \frac{x - x_2 - x_3 + x_4 + x_5 + \frac{x_6}{x^2 + 1}}{x^2 + 1} \\
+ \frac{x_4}{x^2 + 1} + \frac{\xi_1}{x^2 + 1} - \frac{\xi_2}{x^2 + 1}
\end{array} \right\}$$

with $v = (v_1, v_2)^T$ the new control vector. Note that the linearization of $Q$ around the equilibrium yields the linear compensator $LQ$. With respect to the new control vector $v$ in $Q$ there is no obstacle to choose it as in the linear case, eq. (20), that is,

$$\begin{align*}
\tilde{v}_1 &= -4\xi_1 + \tilde{x}_1 + \tilde{v}_1 \\
\tilde{v}_2 &= 4x_2 - 4x_3 + 8\xi_6 + \tilde{v}_2,
\end{align*}$$

with $\tilde{v}_1, \tilde{v}_2$, the new controls.

Figures 3 and 4 illustrate numerical results of the composed system $\Sigma_q \circ LQ$. Here $LQ$ is as described above with $\tilde{v}_1$ and $\tilde{v}_2$ specified in (20) with $\tilde{v}_1 = \tilde{v}_2 = 0$, and the initial condition of $\Sigma_q$ is $x(0) = (0.2, 0.5, 0, 0, 0.6, 0)^T$.

5. Conclusions

The purpose of this paper was to study the disturbance decoupling problem for a nonlinear system in relation to the same problem for its linearization. The main result, see Theorem 3.10, states that under generic conditions - which is a mathematical phrasing for almost always - the problem is solvable in the nonlinear case via dynamic state feedback and only if the linear problem is solvable. Since it is known that if for a linear system the disturbance decoupling problem is solvable, then it is solvable via a (linear) state feedback (cf. [16]), we arrive at the remarkable conclusion that the nonlinear disturbance decoupling problem is solvable via a dynamic state feedback if and only if the linear(ized) disturbance decoupling problem is solvable via a static state feedback.

The above result induces an interesting way of obtaining an approximate solution of the nonlinear disturbance decoupling problem, namely by taking a suitably defined linear dynamic compensator that achieves disturbance decoupling. The dynamic compensator we use is of a specific nature and arises as the linearization of a nonlinear decoupling compensator. One of the specific features of the considered dynamic compensator is the introduction of extra integral actions on a part of the input channels. This explains the use of adding integral action in achieving disturbance attenuation. This idea of providing an approximation for a solution of the nonlinear dynamic disturbance decoupling problem was illustrated on a mathematical simulation example.
References


Figure 1: The output $y_1$ of the system $\Sigma_q$ in closed loop with the linear static state feedback (46).

Figure 2: The output $y_2$ of the system $\Sigma_q$ in closed loop with the linear static state feedback (46).

Figure 3: The output $y_1$ of the system $\Sigma_q \circ LQ$.

Figure 4: The output $y_2$ of the system $\Sigma_q \circ LQ$. 