From dot to ring: the role of friction on the deposition pattern of a drying colloidal suspension droplet

Qingguang Xie\textsuperscript{1,*} and Jens Harting\textsuperscript{2,1,†}
\textsuperscript{1}Department of Applied Physics, Eindhoven University of Technology, P.O. Box 513, 5600MB Eindhoven, The Netherlands
\textsuperscript{2}Helmholtz Institute Erlangen-Nürnberg for Renewable Energy (IEK-11), Forschungszentrum Jülich, Fürtner Str. 248, 90429 Nürnberg, Germany

The deposition of particles on a substrate by drying a colloidal suspension droplet is at the core of applications ranging from traditional printing on paper to printable electronics or photovoltaic devices. The self-pinning induced by the accumulation of particles at the contact line plays an important role in the formation of the deposition. In this paper, we investigate both numerically and theoretically, the effect of friction between the particles and the substrate on the deposition pattern. Without friction, the contact line shows a stick-slip behaviour and a dot-like deposit is left after the droplet is evaporated. By increasing the friction force, we observe a transition from a dot-like to a ring-like deposit. We propose a theoretical model to predict the effective radius of the particle deposition as a function of the friction force. Our theoretical model predicts a critical friction force when the self-pinning happens and the effective radius of deposit increases with increasing friction force, confirmed by our simulation results. Our results can find implications for developing active control strategies for the deposition of drying droplets.

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I. INTRODUCTION

Drying a colloidal suspension droplet is common in many industrial applications, such as inkjet printing [1] and coating processes [2] with applications ranging from the production of newspapers and magazines to printed electronic or even photovoltaic devices. During liquid evaporation, the particles accumulate and finally form a dry solid deposit on the substrate [3–7]. The homogeneity and conformation of the deposition are crucial for the device performance [8, 9]. Therefore, a detailed microscopic understanding and active control strategies for deposition formation process are needed.

The particle deposition pattern is largely affected by the evaporation behaviour of the droplet. Generally, there are two basic modes [10–12] of droplet evaporation: constant contact radius (CR) mode, constant contact angle (CA) mode. In the CR mode, the droplet evaporates with constant radius while its contact angle decreases [10]. In the CA mode, the droplet keeps a constant contact angle but the contact radius decreases [10]. In reality, the contact line mostly shows combinations of the CA and CR modes due to irregular roughness of the substrate, which is called “stick-slip mode” [13, 14]. In the stick-slip mode, the droplet first evaporates in the CR mode until a certain contact angle is reached. Then, the evaporation mode switches to the CA mode [11]. The CA and CR mode can be treated as extreme cases of the stick-slip mode.

Drying of a pure liquid drop on a solid substrate generally shows a stick-slip contact line behaviour [15, 16].

The droplet starts to evaporate with a pinned contact line, followed by a second phase where the contact line depins, the contact radius shrinks and the contact angle remains constant. Deegan [17] observed that the second phase can be absent in drying colloidal drops. The contact line dynamics can be altered by the accumulation of the solute at the contact line. A self-pinning mechanism is proposed: the roughness or chemical heterogeneities of the substrate provide a primary source of contact line pinning, then the accumulation of colloidal particles at the contact line apparently strengthens the pinning, and eventually becomes the dominant contribution to the contact line pinning. The evaporation of a colloidal suspension droplet with pinned contact line introduces an outward capillary flow inside the droplet, transporting the colloidal particles to the edge [3, 18].

The particles accumulate at the contact line, strengthen the pinning of the contact line, and finally form a ring-like deposit known as the “coffee-ring” [3].

Recently, Sangani \textit{et al.} [19] found that the capillary force subjected on particles at the contact line can be sufficiently large to overcome the self-pinning. The capillary force pushes the particles inward, resulting in a dot-like deposit. Weon \textit{et al.} [20] further studied the self-pinning effect of a spreading colloidal droplet. They found that the capillary force is dominant for the self-pinning, while the friction force between particle and substrate is negligible. However, very recently, Wang \textit{et al.} [21] studied the effect of a single colloidal particle on the contact line using the molecular dynamics method. They found that the contact line can experience pinning, depinning and the pinning-depinning transition dependent on the friction force between particle and substrate. Meanwhile, Man and Doi [22] showed that the contact line friction is a key parameter for a ring to mountain-like transition in
the deposition of drying droplets.

In this paper, we numerically study the effect of friction between the particles and the substrate on the deposition using the lattice Boltzmann method. Firstly, we validate the applicability of our model by studying the CR and stick-slip mode of an evaporating pure droplet. We compare the time dependent radial velocity of a drying pure droplet in CR mode with its respective analytical prediction. Then, we investigate the particle deposition obtained from a drying colloidal suspension droplet by varying the friction force. Without friction, the deposition shows a dot-like deposit after the droplet is evaporated. With increasing friction force, the deposition shows a dot-like to ring-like transition, indicating that the particles can introduce self-pinning for a large friction force. In this work, we consider a perfectly flat substrate and demonstrate that the particles with large friction can act as topographical defects, which is consistent with the results in previous work [16]. However, the transport of colloidal particles is usually modeled with a convection-diffusion equation [16, 23], and the interactions between particles and substrate are neglected. Here, we show that the interactions between particle and substrate can play an important role on the deposition pattern. We propose a theoretical model to predict the effective radius of deposition as a function of the friction force, which is in good agreement with our simulation results. Moreover, our theoretical model reveals that self-pinning is determined by the competition between capillary and friction forces.

II. SIMULATION METHOD

We use the lattice Boltzmann method (LBM) which can be treated as an alternative numerical method to describe the dynamics of the fluid. In the limit of small Knudsen and Mach numbers, the Navier-Stokes equations are recovered [24]. In the past two decades, the LBM has demonstrated itself as a powerful tool for numerical simulations of fluid flows [24, 25], and has been extended to simulate multiphase/multicomponent fluids [26, 27] and suspensions of particles of arbitrary shape and wettability [28–31]. The LBM is a local mesoscopic algorithm, allowing for efficient parallel implementation. We review some relevant details in the following and refer the reader to the relevant literature [27, 29, 32, 33] for a detailed description of the method and our implementation.

We utilize the pseudopotential multicomponent LBM of Shan and Chen [26] with a D3Q19 lattice [34]. Here, two fluid components are modeled by following the evolution of each distribution function discretized in space and time according to the lattice Boltzmann equation:

\[ f_i^c(x + \mathbf{e}_i \Delta t, t + \Delta t) = \frac{\Delta t}{\tau_i^c} \left[ f_i^c(x, t) - f_i^{eq}(\rho^c(x, t), \mathbf{u}^c(x, t)) \right] + \mathbf{u}_i^c \cdot \mathbf{v}^c(x, t), \]  

where \( i = 0, \ldots, 18 \). \( f_i^c(x, t) \) are the single-particle distribution functions for fluid component \( c = 1 \) or \( 2 \), and \( \mathbf{e}_i \) is the discrete velocity in the \( i \)th direction. \( \tau_i^c \) is the relaxation time for component \( c \) and determines the viscosity. The macroscopic densities and velocities for each component are defined as \( \rho^c(x, t) = \rho_0 \sum_i f_i^c(x, t) \) where \( \rho_0 \) is a reference density, and \( \mathbf{u}^c(x, t) = \sum_i f_i^c(x, t) \mathbf{e}_i / \rho^c(x, t) \), respectively. Here, \( f_i^{eq} \) is the second-order equilibrium distribution function, defined as

\[ f_i^{eq}(\rho^c, \mathbf{u}^c) = \omega_i \rho^c \left[ 1 + \frac{\mathbf{e}_i \cdot \mathbf{u}^c}{c_s^2} \right] - \frac{(\mathbf{u}^c \cdot \mathbf{u}^c)}{2c_s^2} + \frac{(\mathbf{e}_i \cdot \mathbf{u}^c)^2}{2c_s^2}, \]

where \( \omega_i \) is a coefficient depending on the direction: \( \omega_0 = 1/3 \) for the zero velocity, \( \omega_{1,\ldots,6} = 1/18 \) for the six nearest neighbors and \( \omega_{7,\ldots,18} = 1/36 \) for the nearest neighbors in diagonal direction. \( c_s = \frac{1}{\sqrt{3}} \Delta x / \Delta t \) is the speed of sound.

For convenience we choose the lattice constant \( \Delta x \), the timestep \( \Delta t \), the unit mass \( \rho_0 \) and the relaxation time \( \tau_s \) to be unity, which leads to a kinematic viscosity \( \nu_s = \frac{1}{5} \) in lattice units.

The pseudopotential multicomponent model introduces a mean-field interaction force

\[ \mathbf{F}^c(x, t) = -\Psi^c(x, t) \sum_i \sum_c \omega_i g_{c\bar{c}} \Psi^\bar{c}(x + \mathbf{e}_i, t) \mathbf{e}_i, \]

between fluid components \( c \) and \( \bar{c} \) [26], in which \( g_{c\bar{c}} \) is a coupling constant, eventually leading to a demixing of the fluids. We note \( \gamma_{12} \) as the surface tension of the interface. \( \Psi^c(x, t) \) is an “effective mass”, chosen as the functional form

\[ \Psi^c(x, t) \equiv \Psi(\rho^c(x, t)) = 1 - e^{-\rho^c(x, t)}. \]

This force \( \mathbf{F}^c(x, t) \) is then applied to the component \( c \) by adding a shift \( \Delta \mathbf{u}^c(x, t) = \tau_s \mathbf{F}^c(x, t) / \rho^c(x, t) \) to the velocity \( \mathbf{u}^c(x, t) \) in the equilibrium distribution.

We introduce an interaction force between the fluid and wall, inspired by the work of Huang et al. [35]

\[ \mathbf{F}^c(x) = -g_{w\bar{c}} \Psi^\bar{c}(x) \sum_i \omega_i s(x + \mathbf{e}_i) \mathbf{e}_i, \]

where \( g_{w\bar{c}} \) is a constant. Here, \( s(x + \mathbf{e}_i) = 1 \) if \( x + \mathbf{e}_i \) is a solid lattice site, and \( s(x + \mathbf{e}_i) = 0 \) otherwise.

When the interaction parameter \( g_{c\bar{c}} \) in Eq. (3) is properly chosen [36], a separation of components takes place. Each component separates into a denser majority phase of density \( \rho_{ma} \) and a lighter minority phase of density \( \rho_{mi} \), respectively. The interface is diffusive, which avoids the stress singularity at the moving contact line usually occurring in sharp-interface models.

In order to trigger evaporation, we impose the density of component \( c \) at the boundary sites \( x_H \) to be of constant value \( \rho^c(x_H, t) = \rho_H^c \) by setting the distribution function of component \( c \) to [33]

\[ f_i^c(x_H, t) = f_i^{eq}(\rho_H^c, \mathbf{u}_H^c(x_H, t)), \]

...
in which $\mathbf{u}_p(x_H, t) = 0$. In the case where the set density $\varphi_{mi}$ is lower than the equilibrium minority density $\varphi_{mi}^c$, a density gradient in the vapor phase of component $c$ is formed. This gradient causes component $c$ to diffuse towards the evaporation boundary. We note that our evaporation model is diffusion dominated, which is validated in our previous work [33]. For further discussions on different approaches (such as transfer-rate dominated) to model evaporation, we refer the reader to the relevant literature [16, 37].

The colloidal particles are discretized on the fluid lattice and coupled to the fluid species by means of a modified bounce-back boundary condition as pioneered by Ladd and Aidun [28, 38]. The particles follow classical equations of motion

$$F_p = m \cdot \frac{d\mathbf{u}_p}{dt},$$

in which $F_p$ is the total force imposed on a particle with mass $m$ and $\mathbf{u}_p$ is the velocity of the particle. The trajectory of a colloidal particle is updated using a leap-frog integrator.

We fill the outer shell of the particle with a “virtual” fluid by an amount $\Delta \rho$ [29, 32],

$$\rho_{\text{cirt}}(x, t) = \mathcal{P}^i(x, t) + |\Delta \rho|,$$

$$\rho_{\text{cirm}}(x, t) = \mathcal{P}^2(x, t) - |\Delta \rho|,$$

where $\mathcal{P}^i(x, t)$ and $\mathcal{P}^2(x, t)$ are the averages of the density of neighbouring fluid nodes for component 1 and 2, respectively. The parameter $\Delta \rho$ is called the “particle colour” and dictates the contact angle of the particle. A particle colour $\Delta \rho = 0$ corresponds to a contact angle of $\theta = 90^\circ$, i.e. a neutrally wetting particle.

The momentum exchange between particles and fluid recovers the hydrodynamic forces, such as drag and lift forces. Moreover, our model recovers the lubrication interactions correctly when the distance between two particles is at least one lattice site. If there is less than one lattice site between the particles, a lubrication correction is introduced [28, 39, 40],

$$F_{ij} = -\frac{3\pi \mu a^2}{2} \hat{r}_{ij} \cdot (\mathbf{u}_i - \mathbf{u}_j) \left( \frac{1}{r_{ij}^3} - \frac{1}{\Delta c} \right)$$

where $a$ is the radius of the particle, $\hat{r}_{ij} = \frac{\mathbf{r}_i - \mathbf{r}_j}{r_{ij}}$ is a unit vector pointing from one particle center to the other one and $r_{ij}$ is the distance between particle $i$ and $j$. $\mathbf{u}_i$ and $\mathbf{u}_j$ are particle velocities. $\Delta c$ is a constant and is chosen to be $\Delta c = 2/3$.

To avoid overlapping of particles, we add a Hertz potential between the particles with the following form for two spheres with identical radii $a$ [41]:

$$\phi_H = \begin{cases} K_H (2a - r_{ij})^2 & \text{for } r_{ij} \leq 2a \\ 0, & \text{otherwise} \end{cases}$$

where $K_H$ is the force constant and is chosen to be $K_H = 100$.

For the interactions between particles and a substrate, the lubrication forces between particles and walls are modeled in a similar way as that between particles. Additionally, we implement the Lennard-Jones (LJ) potential between particles and a substrate as

$$\phi_{L,J} = 4\epsilon \left( \frac{\sigma}{r_{ij}} \right)^{12} - \left( \frac{\sigma}{r_{ij}} \right)^6 \text{ for } r_{ij} \leq 2.5a$$

where $\epsilon$ is the depth of the potential well, $\sigma$ is the finite distance at which the inter-particle potential is zero, and $r_{ij}$ is the distance between a particle center and the substrate surface. We set $\sigma$ equal to the particle radius $a$ in all simulations.

To model the tangential friction between particles and a substrate, we apply a constant friction force on the particles as follows:

$$F_f = \frac{-F_f u_{\parallel p}}{|u_{\parallel p}|} \text{ for } r_{iw} \leq a + 1 \text{ & } |F_{\parallel p}| > F_f,$$

where $u_{\parallel p}$ is the component of velocity of the particle parallel to the substrate. Here, $F_{\parallel p}$ is the force acting on the particle due to the particle-particle interaction and particle-fluid interactions in the direction parallel to the substrate. The constant $F_f$ represents the maximum friction. When $|F_{\parallel p}| \leq F_f$ and $r_{iw} \leq a + 1$, the particles get stuck on the substrate. For simplicity, we assume that the static and kinetic friction are equal and the friction force is independent on the normal force exerted on the particle. We note that Eq. (13) satisfies the classical Coulomb law of friction, i.e. the direction of the friction force is opposite to the sliding velocity of the particles and the kinetic friction is independent on the sliding velocity. Similar friction models have been successfully applied to investigate the structure formation of a drying colloidal suspension film [42] and granular dynamics [43]. Recently, Guo et al. experimentally measured the friction between a nanoparticle and a substrate and demonstrated that the sliding kinetic friction is constant [44].

III. RESULTS AND DISCUSSION

A. Pure droplet

We start out to simulate the evaporation of a pure droplet sitting on a solid substrate, as illustrated in Fig. 1. The simulations utilize a system size of $256 \times 256 \times 144$ and the substrate is chemically patterned with a variable wettability: a superhydrophilic circle ($\theta \approx 0^\circ$) of radius $R_s = 115$ is located at the center surrounded by a superhydrophobic area ($\theta \approx 180^\circ$). We initialize the droplet with a contact radius $R_0 = 115$ of initial maximal height $h_0 = 100$, and densities $\rho_{ma}^c = \rho_{mi}^c = \rho_{mi}^c = 0.70$, $\rho_{mi}^c = \rho_{mi}^c = 0.04$. The initial contact angle of the droplet is $\theta_0 \approx 82^\circ$. After equilibration, we apply
We place a chemically patterned substrate with variable wettability at the bottom, while the boundaries normal to the substrate are periodic. After equilibration, we apply evaporation boundary conditions at the sides and the top of the system (denoted by the dashed lines).

Fig. 2 shows the evolution of the normalized contact radius \( R_d/R_0 \), droplet height \( h/h_0 \) and contact angle \( \theta/\theta_0 \) versus time. The time is normalized with the diffusivity of the fluid, \( D \approx 0.117 \) [33]. During the evaporation, the contact angle of the droplet decreases while the contact radius keeps constant. This is the so-called CA mode [10], resulting from the contact line pinning caused by contact angle heterogeneities of the substrate. The droplet height and contact angle decrease linearly.

The evaporation of a droplet with pinned contact line introduces an outward capillary flow inside the droplet [3, 45]. Following the principle of mass conservation [3, 45, 46] and assuming an infinite size of the system, the evolution of the droplet height in the limit of small contact angles is obtained as

\[
h(0, t) = \frac{8D\Delta \rho}{\pi \rho R_d} \left( t - t_e \right),
\]

(14)

with \( t_e \) denoting the total life time of the droplet. We note that Eq. (14) predicts that the droplet height decreases linearly with time, which is consistent with our simulation results shown in Fig. 2. Using the lubrication approximation [46], one can estimate the radial velocity near the substrate as

\[
u(r, z, t) = \frac{3}{h^2(r, t)} \bar{u} \left( h(r, t)z - \frac{1}{2}z^2 \right),
\]

(15)

where \( \bar{u} \) is the average radial velocity and is given as

\[
\bar{u} = \frac{R_d^3}{4r} \left[ \frac{1}{\sqrt{R_d^2 - r^2}} - \frac{1}{R_d^2} \left( R_d^2 - r^2 \right) \right] \frac{1}{t_e - t}.
\]

(16)

The term \( \frac{1}{t_e - t} \) in Eq. (16) predicts that the radial velocity diverges near the end of the lifetime of the droplet, which is confirmed by experiments [46]. For a detailed derivation, we refer the reader to the Appendix A.

In the simulations, we measure the radial velocity at position \( r = 100, z = 6 \) and compare the simulation data with the theoretical analysis Eq. (15) in Fig. 3. The total lifetime of the droplet \( t_e \) is determined from a linear fit of the time evolution of the droplet height (dashed line.
shown in Fig. 2). The radial velocity increases with time and shows a rapid increase at later times, which agrees well with the theoretical prediction Eq. (15).

![Graph](image)

**Fig. 4:** The evolution of the contact radius $R_d/R_0$, the droplet height $h/h_0$, and the contact angle $\theta/\theta_0$ of a pure droplet for $\rho_f = 0.01$. The contact line shows a stick-slip behavior: firstly, the contact angle decreases while the contact radius keeps constant, followed by a phase where the contact angle keeps constant while the contact radius decreases.

Next, we investigate the stick-slip mode of a drying droplet by creating a hydrophilic circular area with contact angle $\approx 56^\circ$ located at the center of the substrate and the remaining area is hydrophobic of contact angle $\approx 124^\circ$. We place a droplet with an initial radius $R_0 = 115$ and an initial height $h_0 = 115$, therefore, the initial contact angle is $\theta_0 = 90^\circ$.

Fig. 4 shows the time evolution of the contact radius $R_d/R_0$, the droplet height $h/h_0$, and the contact angle $\theta/\theta_0$ of the evaporating droplet. In the beginning, the contact angle decreases while the contact radius keeps constant. This indicates that the contact line is pinned at the border between the hydrophilic and the hydrophobic area due to the wettability heterogeneities of the substrate. When the contact angles reaches $\approx 56^\circ$, it stays constant, while the contact radius begins to decrease, indicating that a depinning of the contact line occurs. The droplet height decreases throughout the drying time of the droplet, but with a faster decreasing rate in the pinned phase than in the unpinned phase, which is consistent with experimental results [12]. Therefore, we demonstrate that our system is able to reproduce the stick-slip mode accurately by utilizing the patterned wettability of the substrate. We note that the stick-slip motion can also be triggered by the topological features on the substrate [16]. For simplicity, we initialize the substrate with patterned wettability to introduce the stick-slip mode.

**B. Colloidal suspension droplet**

In the following we investigate the evaporation of a colloidal suspension droplet. We initialize the droplet with a particle volume concentration $\Phi \approx 0.89\%$ which corresponds to 500 particles of radius $a = 3$. The particles are slightly hydrophilic with a contact angle $\theta_p \approx 75^\circ$ and the substrate has a hydrophilic circular area with a contact angle $\approx 56^\circ$ located at the center surrounded by a hydrophobic area of contact angle $\approx 124^\circ$. We note that the pure droplet shows a stick-slip drying mode on this specific substrate. We apply a constant friction force with magnitude $F_f$ following Eq. (13) between particles and substrate. Then, we let the system equilibrate and apply evaporation boundary conditions on the sides and the top of the system.

![Graph](image)

**Fig. 5:** The evolution of the contact radius $R_d/R_0$, the droplet height $h/h_0$ and the contact angle $\theta/\theta_0$ of a colloidal suspension droplet without friction.

Firstly, we set the friction force to zero and investigate the contact line dynamics. Fig. 5 shows that the contact line of the colloidal suspension droplet initially follows the constant radius mode: the contact radius keeps constant and the contact angle decreases. When the contact angle reaches $\theta \approx 56^\circ$, it slightly increases and then stays constant. Meanwhile, the contact radius begins to decrease, indicating that the contact line depins. The droplet height keeps decreasing in the whole process of drying, which is also observed in the stick-slip mode of a drying pure droplet. We note that the contact angle at the slip stage ($\theta \approx 75^\circ$) is larger than the one observed for a pure droplet ($\theta \approx 56^\circ$) (Fig. 4). The reason is that the particles accumulate at the contact line and serve as a new substrate. The contact line is pinned at the particle surface. Thus, the contact angle of the droplet is determined by the particle wettability, i.e. $\theta_p \approx 75^\circ$.

We show the corresponding drying process of the col-
FIG. 6: Snapshots of a drying colloidal suspension droplet without friction. The particles are transported to the contact angle, and are pushed inwardly by the capillary force. Finally, the particles form a dot-like pattern.

FIG. 7: The evolution of the contact radius $R_d/R_0$, the droplet height $h/h_0$ and the contact angle $\theta/\theta_0$ of an evaporating colloidal suspension droplet for a large friction force $F_f/a\gamma_{12} = 0.07$.

In order to investigate the influence of friction between particles and substrate, we set $F_f/(a\gamma_{12}) = 0.07$. Fig. 7 shows the time evolution of the contact radius $R_d/R_0$, the droplet height $h/h_0$ and the contact angle $\theta/\theta_0$ of the evaporating colloidal suspension droplet. At first, the contact radius keeps constant and the contact angle decreases, which is similar to the behavior of a drying colloidal suspension droplet without friction, as shown in Fig. 5. Interestingly, when the contact angle decreases to $\approx 56^\circ$, there is only a slight decrease of the contact radius, in contrast to the continuous decrease of the contact radius observed in the drying of a pure droplet and the drying of a colloidal suspension droplet without friction. Moreover, the contact angle keeps decreasing, which is similar to the constant radius mode observed for a drying pure droplet (Fig. 2).

To understand the behavior of the contact line, we show the drying process of the colloidal suspension droplet obtained in our simulations in Fig. 8. The particles are initially randomly distributed in the droplet and some are adsorbed at the droplet interface after equilibration. During evaporation in CR mode, the particles are transported to the contact line and accumulate there (Fig. 6a). When the contact angle reaches a critical value, the contact line depins, recedes and pulls the particles inward through a capillary force. However, the particles are stuck at the substrate due to the large friction and introduce a pinning of the contact line (Fig. 6b). The droplet further dries and more particles are transported to the contact line forming a porous structure. The interface recedes through the porous structure, resulting in a slight decrease of the contact radius, as shown in Fig. 7. Near the end of the lifetime of the droplet, the contact angle decreases and the droplet flattens. This thin flat film breaks close to the ring (Fig. 8c), is dried rapidly, and finally a ring-like deposit is left (Fig. 8d).

We note that the magnitude of the friction force $F_f/a\gamma_{12} = 0.07$ is chosen to be of the same order as the magnitude of the capillary force of the particles at the contact line, which has been calculated in the work of Sangani et al. [19]. In reality, the friction force arises from the normal load on the particles and the surface roughness of both, particles and substrate. For a possible experimental validation of our results, we refer to recent atomic force microscopy measurements of the friction force between particles and a substrate [44].

C. Theoretical model and discussion

We propose a simple theoretical model to consider the effect of the friction force on the effective radius of de-
position. In the case of a dilute colloidal suspension droplet, we assume a uniform distribution of particles in the droplet during evaporation. Thus, the particle number density $\rho_n = N_P/V_0$, where $N_P$ is the total number of particles, and $V_0$ is the initial droplet volume, keeps constant inside the evaporating droplet and the remaining particles are transported to the contact line. For simplicity, we also assume that the particle deposition at the contact line is a monolayer and the particles are uniformly distributed along the contact line.

Fig. 9 depicts the aggregation of particles at the contact line. The outermost particle deforms the interface, thus a capillary force $F_c$ rises up and acts on the particle pointing inwards. Meanwhile, the outermost particle and its neighbouring particles along the radial direction experience a friction force $F_f$, which resists the inward capillary force. We define the outermost particle and its neighbouring particles along the radial direction as a radial particle cluster. The contact line depins when the capillary force is larger than the total friction force $|F_c| \geq |F_f|$ which is given by $F_f = N_{\text{cluster}}F_f$. Here, $N_{\text{cluster}}$ is the number of particles in the radial cluster.

During the constant radius phase of evaporation, the contact angle of the droplet decreases and the capillary force acting on the outermost particles increases. Meanwhile, more particles are transported to the contact line and accumulate there resulting in an increase of the total friction force. Therefore, the competition between capillary force and total friction force determines the moment of depinning of the contact line and the deposition pattern.

If the accumulated particles are not able to pin the contact line at the end of the constant radius phase $(|F_c| \geq |F_f^{\text{total}}|)$, the contact line depins and shrinks while the contact angle keeps constant. We assume the capillary force acting on the outermost particles to be constant. However, the total friction force increases due to the continuous accumulation of more particles at the contact line. When the total friction force becomes larger than the capillary force, the contact line is pinned at a position with a certain distance $R_p$ to the center of the hydrophilic area. The droplet continues to dry in constant radius mode and leaves a deposition pattern with an effective radius $R_p$ determined by

$$N_{\text{cluster}}F_f = F_c.$$  

(17)

In order to derive an expression for $N_{\text{cluster}}$, we assume that the droplet keeps a spherical cap with volume $V_p$ when the contact line is pinned at the position with distance $R_p$ to the center. The total number of particles that are transported to the contact line is given by $V_0 - V_p \rho_n$. Based on a geometric analysis, we obtain

$$V_0 = \frac{\pi h_0}{6} (3R_0^2 + h_0^2),$$

$$V_p = \frac{\pi h_p}{6} (3R_p^2 + h_p^2),$$

(18)

where $h_p$ is the maximal droplet height when the self-pinning happens. With the assumption that the particles form a circular ring (outer radius $R_p$, inner radius $r_p$) at the contact line, we obtain its area as

$$\pi (R_p^2 - r_p^2) = (V_0 - V_p) \rho_n \pi a^2.$$  

(19)

The average number of particles in the radial particle cluster is then

$$N_{\text{cluster}} = \frac{R_p - r_p}{2a}.$$  

(20)
Using Eq. (19) and Eq. (20), we can write Eq. (17) as

\[ R_p - \frac{\sqrt{R_p^2 - (V_0 - V_p)\rho_n a^2}}{2a} F_f = F_c. \]  
(21)

Based on Eq. (21), we note that the number of particles \((V_0 - V_p)\rho_n\) at the contact line directly affects the pinning of the contact line, which is consistent with the conclusion drawn in the work of Weon et al. [20] that the particle packing fraction at the contact line plays an important role in self-pinning.

Here, we determine a critical value of the friction force \(F_{cr}^f\) below which the contact line depins at the end of the constant radius phase. At the end of the constant radius phase, the number of particles that are transported to the contact line is \((V_0 - V_1)\rho_n\), where \(V_1 = \frac{\pi R_p (1 - \cos \theta_d)}{6 \sin \theta_d} \left[ 3 R_0^2 + \left( \frac{R_p (1 - \cos \theta_d)}{\sin \theta_d} \right)^2 \right]\). We then obtain the critical value of the friction force as

\[ F_{cr}^f = \frac{2a F_c}{R_0 - \sqrt{R_0^2 - (V_0 - V_1)\rho_n a^2}}. \]  
(22)

If the friction force \(F_f < F_{cr}^f\), the contact line depins and follows a constant angle mode. The contact angle of the droplet is \(\theta_d\) and we can write \(h_p = \frac{R_p}{\sin \theta_d} - \frac{R_p}{\tan \theta_d} = \frac{R_p (1 - \cos \theta_d)}{\sin \theta_d}\). Inserting Eq. (18) into Eq. (21) and after some manipulations we obtain

\[ R_p - \frac{\sqrt{R_p^2 + \frac{\pi \rho_n a^2 (1 - \cos \theta_d)}{6 \sin \theta_d} \left( 3 + \frac{(1 - \cos \theta_d)^2}{(\sin \theta_d)^2} \right)}}{2a} F_f = 2a F_c/F_f. \]  
(23)

We note that there should be a minimal radius of deposition where all particles are located around the center of the substrate. This minimal radius can be obtained as \(R_{cmi} = \sqrt{N_p a^2} \approx 67\) and allows to estimate the minimal friction force \(F_{cmi}^f\) below which particles will always be located around the center as

\[ F_c/F_{cmi}^f = R_{cmi}^f/2a. \]  
(24)

To validate our theoretical analysis, we carry out simulations with different friction forces in the range \(0 \leq F_f/a\gamma_{12} \leq 0.07\). We define the effective radius \(R_p\) of deposition in a way such that a circle of radius \(R_p\) covers 95% of all particles in the system. We solve Eq. (22), Eq. (23) and Eq. (24) with the simulation parameters \(N_p = 500\), \(a = 3\), \(\theta_d = 56^\circ\), \(R_0 = 115\), \(h_0 = 115\). The capillary force is assumed to be constant and we estimate it as \(F_c/a\gamma_{12} \approx 0.068\) [19].

From Eq. (22) and Eq. (24) we obtain the critical friction force \(F_{cmi}^f/a\gamma_{12} \approx 0.034\) and the minimum friction force \(F_{cmi}^f/a\gamma_{12} \approx 0.0064\), respectively. In Fig. 10 we compare the simulation results (symbols) with our theoretical analysis (solid lines). The simulations show that the effective radius of deposition \(R_p\) keeps constant until the friction force reaches the minimal friction force \(F_{cmi}^f\). Then, the effective radius increases for small friction forces, and finally reaches a plateau for large friction forces which is in a good agreement with our theoretical analysis. We note that the theoretical analysis predicts a lower value of the effective radius than simulation results at \(F_f/a\gamma_{12} \approx 0.022\). A possible interpretation is that near the end of the lifetime of the droplet, while the particles are being pulled inwards, the droplet flattens, breaks up and dries quickly. Therefore, the capillary force vanishes and the particles stop moving, resulting a slightly higher effective radius than expected from the theoretical analysis.

We note that our theoretical analysis is only valid when the particles form a monolayer at the contact line, as observed in the experiments by Sangani at al. [19]. However, the multilayer structures are also observed in evaporation-driven deposition [46, 47]. Our theory can be extended to suit for multilayer cases with the structure of the multilayer being provided, such as the thickness of the multilayer along the radial direction.

**IV. CONCLUSION**

We investigated the deposition of a drying droplet on a chemically patterned substrate and demonstrate that we are able to control the wettability of the substrate accurately to reproduce the constant radius mode and the stick-slip mode. When the droplet evaporates with
a pinned contact angle, the radial velocity inside the droplet diverges near the end of the droplet lifetime, which is in a good agreement with theory and experiment.

We then studied the effect of friction between the particles and the substrate on the contact line dynamics and the final deposition pattern in drying colloidal suspension droplets. Without friction, the contact line shows a stick-slip drying mode similar to a drying pure droplet. Interestingly, due to the pinning of the contact line on the particles, the droplet in the slip stage has a larger contact angle than that of a pure droplet. Moreover, the particles follow the receding contact line and finally form a dot-like deposit.

With increasing the friction force, the drying suspension droplet shows a transition from stick-slip mode to constant radius mode. With a large friction force, the particles accumulate at the contact line, introducing self-pinning. Thus, more particles are transported to the contact line and form porous structures. Surprisingly, we find that the contact line recedes through the pores, resulting in a slight decrease of the contact radius.

Moreover, we propose a theoretical model for the effective radius of deposition as a function of the friction force. Our model predicts a critical friction force at which the self-pinning happens and a decreasing effective radius of the deposit with decreasing friction. Additionally, the effective radius of the particle deposit keeps constant for all friction forces smaller than a minimal friction. We carried out simulations and find good agreement between the simulation results and our theoretical predictions. Our results contribute to the fundamental understanding of self-pinning phenomena and can find implications for developing active control strategies for the deposition of drying droplets.

Finally, we note that after the pioneering work of Deegan et al. on the coffee-ring effect [3], the understanding of the macroscopic mass and momentum transport phenomena in drying droplets is relatively well developed [3, 12, 45, 48]. However, the interfacial phenomena, contact-line and deposition processes in drying colloidal suspension droplets are not nearly as well understood [5]. The available theoretical models usually apply a convection-diffusion equation governing the transport of colloidal particles [16, 49], which cannot account for phenomena at the scale of individual particles (e.g., contact-line pinning on particles, particle-particle interactions, particle-substrate interactions). We perform lattice Boltzmann simulations with fully resolved colloidal particles. Thus, we are able to fully resolve the information at the scale of individual particles and can gain more insight in the deposition process.

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Appendix A: Derivation of the radial fluid velocity in a drying droplet with pinned contact line

The average radial velocity can be derived assuming mass conservation [46]. We consider a small volume $\Delta v$. The mass conservation law of this controlled volume $\Delta v$ can be written as

$$\frac{\partial h(r,t)}{\partial t} = -\frac{1}{r} \frac{\partial}{\partial r} Q(r,t) - \frac{1}{\rho} J(r,t), \quad (A1)$$

where $h(r,t)$ is the local height of the droplet surface, $t$ is the time, $r$ is the radial coordinate, $Q(r,t)$ is the volume flow, $\rho$ is the liquid density, and $J(r,t)$ is the evaporative flux.

In the limit of small contact angles, the evaporative flux can be written as $[3, 45, 46]$ $J(r) = \frac{2}{\pi} \frac{D\Delta \rho}{\sqrt{R_d^2 - r^2}}, \quad (A2)$

with $D$ the diffusion constant for the droplet’s liquid in another fluid, and $\Delta \rho$ the difference between the liquid density just above the drop surface and the liquid density located far away from its surface. Here, we assume that the droplet has spherical cap shape and we obtain $h(r,t) = \frac{R_d^2 - r^2}{R_d^2} h(0,t)$. We note that LBM can recover Navier-stokes equation [24], thus we are able to simulate a droplet with larger contact angle and a spherical-cap shape, which goes beyond the limitation of lubrication theory based models of drying droplets [50], where an assumption of a flat droplet is required.

Following global mass conservation, the change in droplet volume

$$\frac{dV}{dt} = \frac{d}{dt} \int_0^{R_d} h(r,t) 2\pi r dr = \frac{\pi R_d^2}{2} \frac{dh(0,t)}{dt} \quad (A3)$$

should be equal to the total amount of evaporated liquid

$$\frac{dV}{dt} = -\frac{1}{\rho} \int_0^{R_d} J(r) 2\pi r dr = -\frac{4R_d D\Delta \rho}{\rho} \quad (A4)$$

Comparing Eq. (A3), and Eq. (A4), we obtain

$$\frac{dh(0,t)}{dt} = -\frac{8D\Delta \rho}{\pi \rho R_d} \quad (A5)$$

The evolution of the droplet height is

$$h(0,t) = \frac{8D\Delta \rho}{\pi \rho R_d} (t_e - t), \quad (A6)$$
with \( t_e \) being the total life time of the droplet. We note that Eq. (A6) predicts that the droplet height decreases linearly with time.

Inserting Eq. (A6) and Eq. (A2) into Eq. (A1) and after some manipulations we obtain

\[
Q(r) = \frac{2D\Delta \rho}{\pi \rho} \left[ \frac{1}{\sqrt{R_d^2 - r^2}} - \frac{1}{R_d^2} \right] \left( R_d^2 - r^2 \right). \tag{A7}
\]

The volume flow \( Q(r) = rh \bar{u} \), where \( \bar{u} \) is the height averaged radial velocity. Using Eq. (A7) and Eq. (A6), it can be derived as

\[
\bar{u} = \frac{Q}{rh(r,t)} = \frac{2R_d^2D\Delta \rho}{\pi \rho r h(0,t)} \left[ \frac{1}{\sqrt{R_d^2 - r^2}} - \frac{1}{R_d^2} \right]. \tag{A8}
\]

Following an approach proposed by Marin et al. [46] based on the lubrication approximation, we can then estimate the radial velocity as

\[
u(r,z,t) = \frac{3}{h^2(r,t)} \bar{u}\left(h(r,t)z - \frac{1}{2}z^2\right). \tag{A9}
\]


